



Contributing to society with our 'Moving it. Stopping it.' technologies

Nabtesco manufactures products which are used in everyday life. Our high-accuracy components are essential for moving objects; they may be rarely visible, but are the foundation of everyday objects that you see moving and wonder how. Nabtesco's technologies are found throughout objects that move and stop people's lives.



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Who is Nabtesco?

The key words for Nabtesco are 'motion control'. We use our strengths in the fields of component and systems technologies to develop highly creative products. Through the Nabtesco Group as a whole, we can also utilize our advantage of expertise to maximum effect in order to further enhance these strengths.

In the air, on land and at sea, we have a leading share in various fields of both international and domestic markets.

Nabtesco will continue to evolve by utilizing its strengths in many fields and by exploring the possibilities of the future.



Teijin Seiki Co., Ltd. Established 1944

Business Merger in 2003

Motion control



April 2002 Initiation of hydraulic equipment business alliance October 2003 Business merger

The business alliance between Teijin Seiki and NABCO on hydraulic equipment projects was the beginning of a mutual confirmation by the companies of the other's product configuration, core technologies, corporate strategies and corporate culture. This led to a common recognition that a business merger would be an extremely effective means of increasing corporate value and achieving long-term development. Based on this mutual judgment, in 2003 an equity transfer was conducted to establish Nabtesco as a pure holding company, with both firms as wholly owned subsidiaries. After a year of preparation, both companies were absorbed and amalgamated by means of a short form merger, and Nabtesco was transitioned to an operating holding company.

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The RA Series includes gearheads for high precision control of ATC magazines, ATC arms, APC, and turret drives of lathe machining centers.



RA Series characteristics and structure

High reliability
High rigidity
High precision
High torque

The double-ended support design and unique pin gear mechanism provide the following advantages:

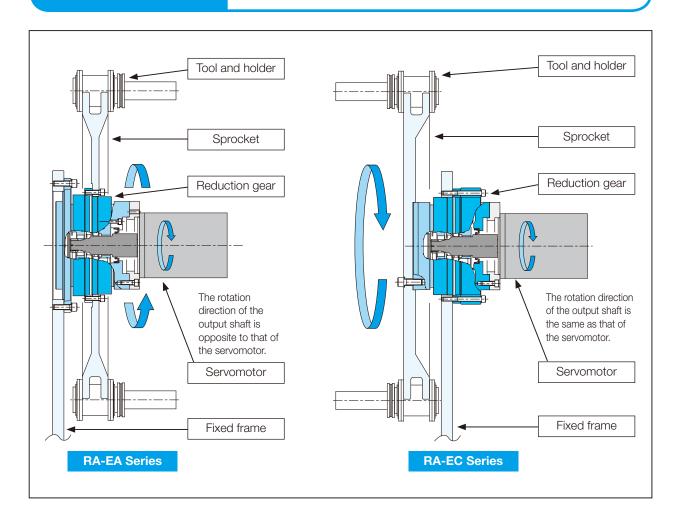
- (1) Capable of high shock load 5 times the rated torque
- (2) High torsional rigidity
- (3) Small backlash [1 arc.min]
- (4) High torque density (capable of high torque with downsized gear)

Heavy load support

A set of internal main bearings (large angular ball bearings) enables complete support of heavy external loads.

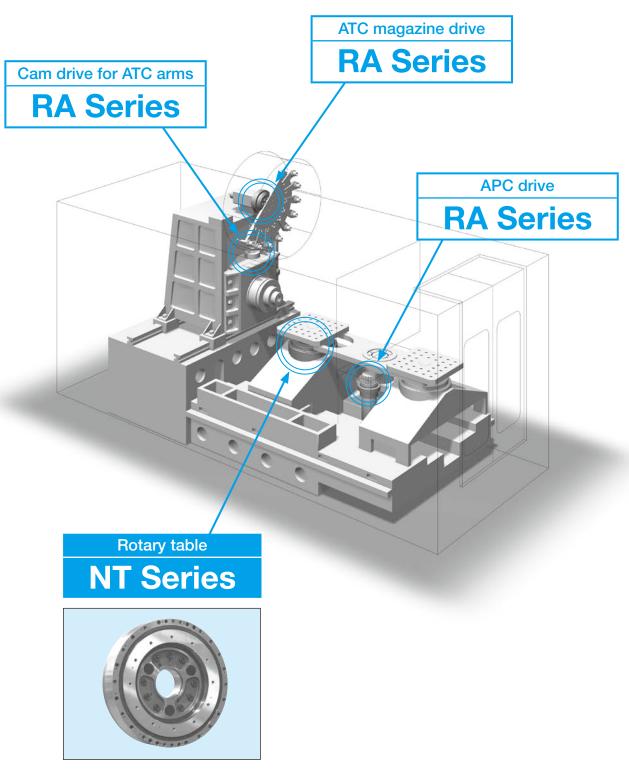
Two benefits due to the above features

- 1. Compact design with a reduced number of parts
- 2. Reduced man-hours (for design, assembly, and adjustment)



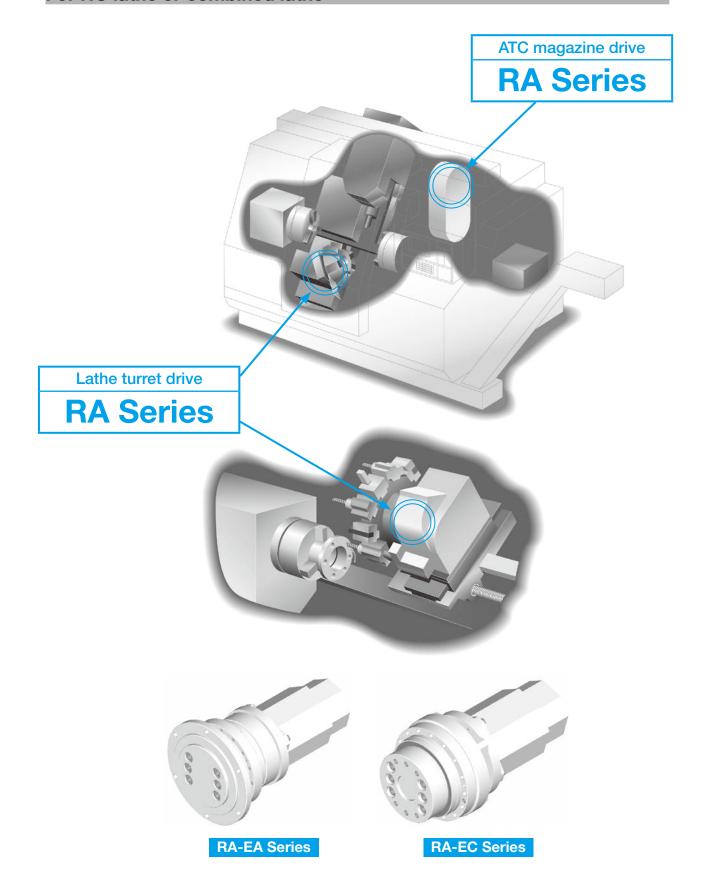
Examples of uses for the RA Series

For machining center



Contact us for NT Series rotary tables.

For NC lathe or combined lathe



Principle of operation

- Rotation of the servomotor is transmitted through the input gear to the spur gears, and the speed is reduced according to the gear ratio between the input gear and the spur gears. <Fig. 1>
- Since the crankshafts are directly connected to the spur gears, they have the same rotational speed as the spur gears. <Fig. 1>
- Two RV gears are mounted around the needle bearings on the eccentric section of the crankshaft. (In order to balance the equal amount of force, two RV gears are mounted.) <Fig. 2>
- 4. When the crankshafts rotate, the two RV gears mounted on the eccentric sections also revolve eccentrically around the input axis (crank movement). <Fig. 2>
- 5. Pins are arrayed in a constant pitch in the grooves inside the case. The number of pins is just one larger than the number of RV gear teeth. <Fig. 3>
- 6. As the crankshafts revolve one complete rotation, the RV gears revolve eccentrically one pitch of a pin (crank movement), with all the RV teeth in contact with all of the pins. As a result, 1 RV gear tooth moves in the opposite direction of the crankshaft rotation. <Fig. 3>
- 7. The rotation is then output to the shaft (output shaft) via the crankshaft so that the crankshaft rotation speed can be reduced in proportion to the number of pins. <Fig. 3>
- The total reduction ratio is the product of the first reduction ratio multiplied by the second reduction ratio.

Fig. 1. First reduction section

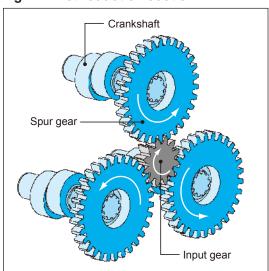


Fig. 2. Crankshaft section

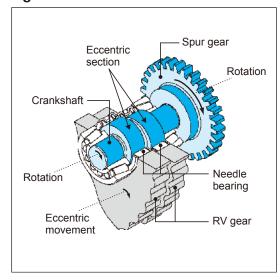
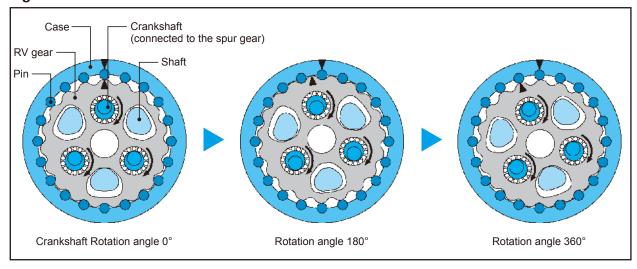
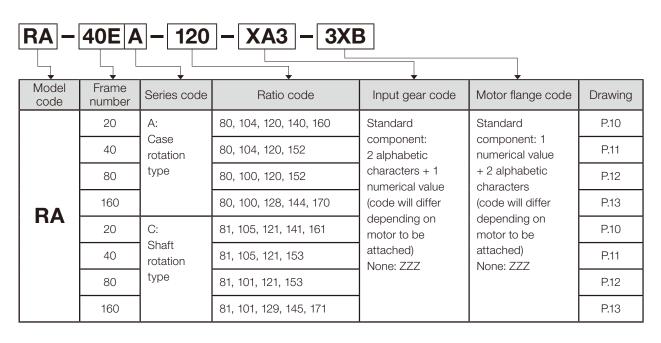


Fig. 3. Second reduction section

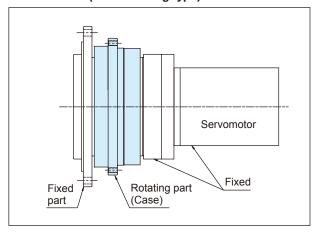


RA Series model code

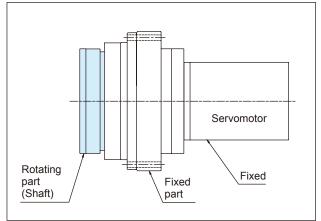
Product code



RA-EA Series (Case rotating type)



RA-EC Series (Shaft rotating type)



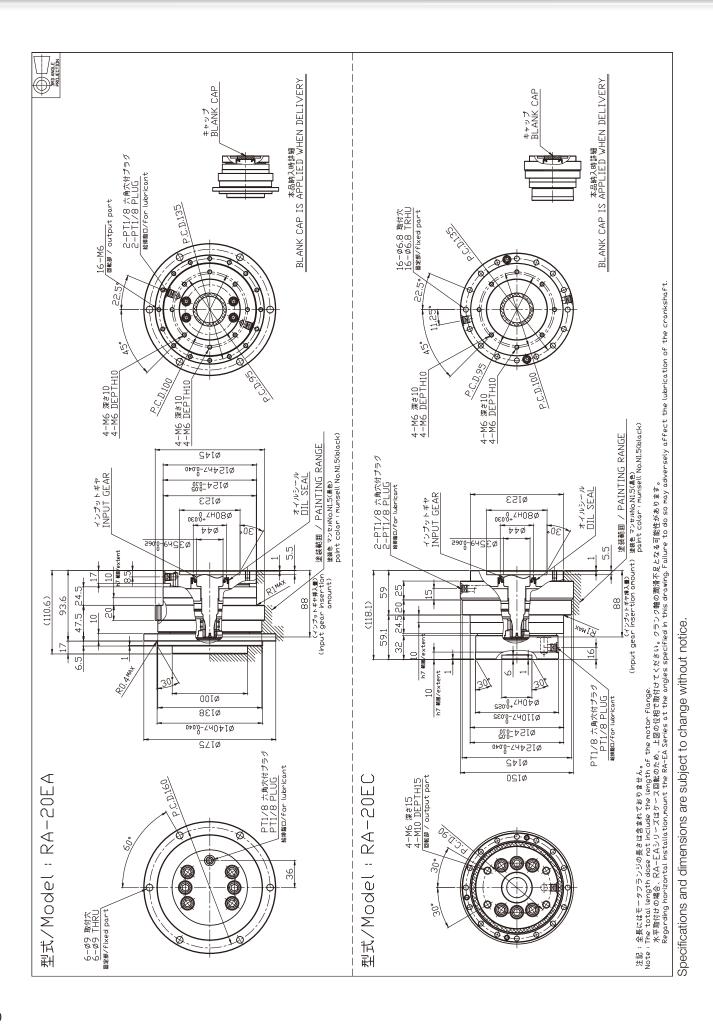
Rating table

Model			R Speed ratio			To Rated torque	No Rated output speed	K Rated life h	Ts1 Allowable acceleration/ deceleration torque
RA-EA Series						INIII	тріпі	- 11	INIII
RA-20EA	80	104	120	140	160	167	15	6000	412
RA-40EA	80	104	120	152		412	15	6000	1029
RA-80EA	80	100	120	152		784	15	6000	1960
RA-160EA	80	100	128	144	170	1568	15	6000	3920
RA-EC Series									
RA-20EC	81	105	121	141	161	167	15	6000	412
RA-40EC	81	105	121	153		412	15	6000	1029
RA-80EC	81	101	121	153		784	15	6000	1960
RA-160EC	81	101	129	145	171	1568	15	6000	3920

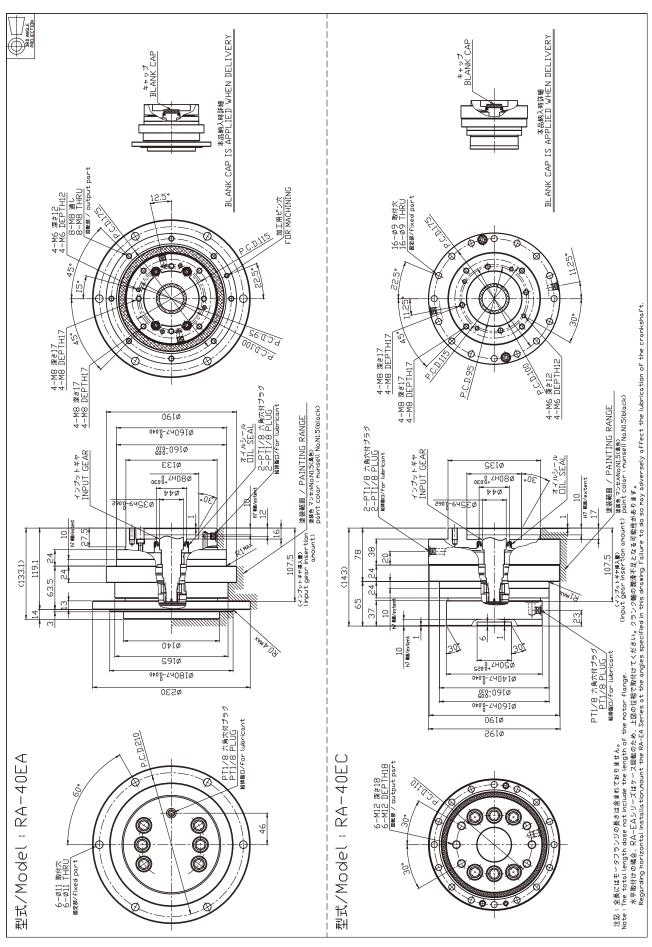
- Note: 1. The rating table shows the specification values of each individual reduction gear.
 - 2. The allowable output speed may be limited by heat depending on the operating rate. Make sure that the surface temperature of the reduction gear does not exceed 60°C during use.
 - 3. For the inertia moment of the reduction gears, refer to the Product Summary Sheet.
 - 4. When the reduction gear is used at low temperatures, there will be a larger no-load running torque. Note this characteristic when selecting a motor. (Refer to "Low temperature characteristics" on page 32)
 - 5. The allowable moment will differ depending on the thrust load. Check the allowable moment diagram (p. 31).
 - 6. For the moment rigidity and torsional rigidity, refer to the calculation of tilt angle and the torsion angle (p. 33).
 - 7. The rated torque is the value that produces the rated service life based on operation at the rated output speed; it does not indicate the maximum load. Refer to "Glossary" (p.21) and "Product selection flowchart" (p.22).
 - 8. The specifications above are based on Nabtesco evaluation methods; this product should only be used after confirming that it is appropriate for the operating conditions of your system.

						Capa	city of main be	earing	
Ts 2	Ns o	Ns1				Mo1	Mo2	Wr	
Momentary maximum allowable torque	Allowable output speed [Duty ratio: 100%]	Allowable output speed [Duty ratio: 40%]	Backlash	Lost motion	Startup efficiency (Representative values)	Allowable moment	Momentary maximum allowable moment	Allowable radial load	Mass
	Note 2	Note 2				Note 5			
Nm	rpm	rpm	arc.min.	arc.min.	%	Nm	Nm	N	kg
833	45	75	1.0	1.0	75	882	1764	7,255	10
2058	42	70	1.0	1.0	70	1666	3332	11,594	18.5
3920	42	70	1.0	1.0	75	2156	4312	12,988	28
7840	27	45	1.0	1.0	75	3920	7840	18,587	58
833	45	75	1.0	1.0	75	882	1764	7,255	9.5
2058	42	70	1.0	1.0	70	1666	3332	11,594	20
3920	42	70	1.0	1.0	75	2156	4312	12,988	27
7840	27	45	1.0	1.0	75	3920	7840	18,587	59

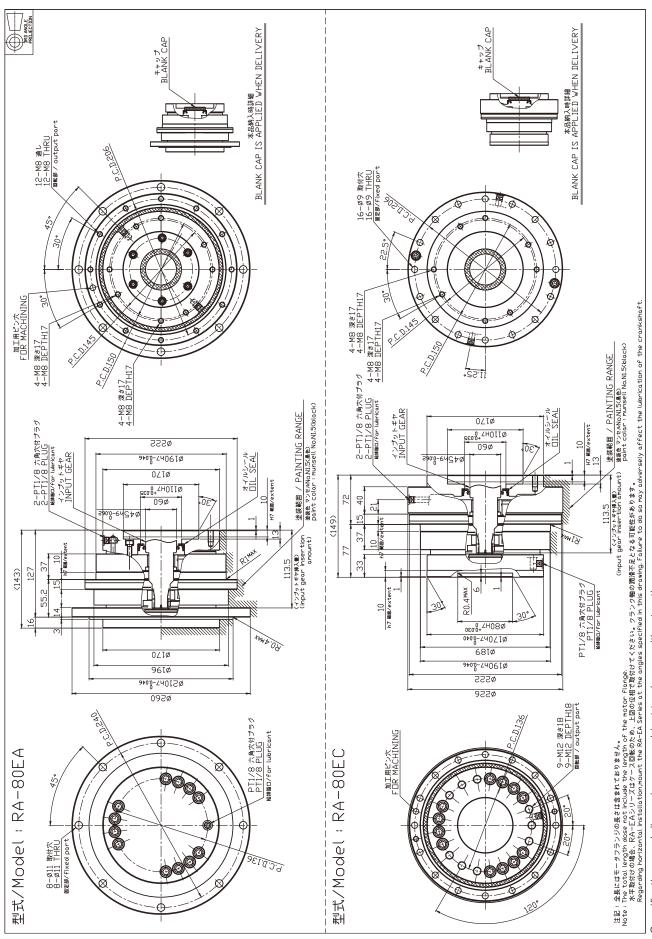
External dimensions Reduction gear main unit



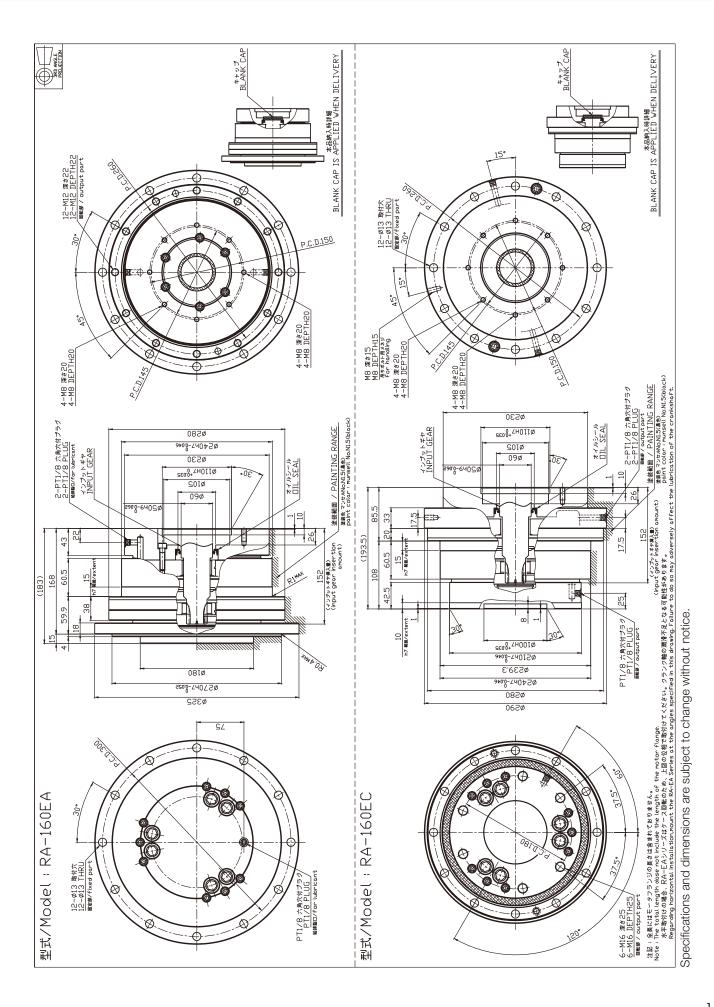
- 10



Specifications and dimensions are subject to change without notice.

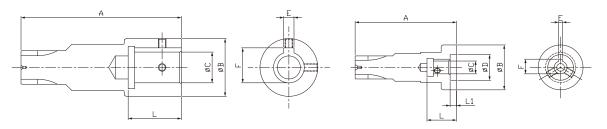


Specifications and dimensions are subject to change without notice.



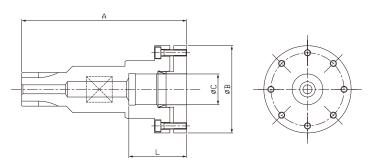
External dimensions Input gear

Straight shaft (with key)



Model	Spee	d ratio	Input gear code	Order item number					Input gea	ar dimens	ons (mm)						
Model	EA	EC	input gear code		Α	ØB	(ØC .	ØD	L	L1		E		F		
	80	81	XS2	21RA107-*													
	104	105	XA2	21RA107A*	70	0.5	401.17	+0.015	00	00	_	0	0.0405		+0.2		
	120	121 141	XB2 XC2	21RA107B* 21RA107C*	79	35	10H7	0	20	23	5	3	±0.0125	11.4	0		
	160	161	XD2	21RA107D*													
	80	81	BL2	21RA107BL*													
	104	105	BM2	21RA107BM*				+0.030							+0.2		
	120	121	BN2	21RA107BN*	94.5	35	14	+0.030	-	21.5	-	5	±0.015	16.3	0		
	140	141	BP2	21RA107BP*				10.012							"		
	160	161 81	BQ2 XK2	21RA107BQ* 21RA107K*													
	104	105	XL2	21RA107L*											l		
RA-20E	120	121	XM2	21RA107M*	135.5	42	24H7	+0.021	-	47.5	-	8	±0.018	27.3	+0.2		
	140	141	XN2	21RA107N*				0							0		
	160	161	XP2	21RA107P*													
	80	81	XQ2	21RA107Q*													
	104	105 121	XR2 XT2	21RA107R* 21RA107T*	113	35	19	+0.036	_	35	_	6	±0.015	21.8	+0.2		
	140	141	XU2	21RA107U*	110	00	13	+0.015		00		0	10.010	21.0	0		
	160	161	XW2	21RA107W*													
	80	81	XX2	21RA107X*													
	104	105	XY2	21RA107Y*	465			+0.036		,_		_			+0.2		
	120	121	XZ2	21RA107Z*	138	42	24	+0.015	-	45	-	8	±0.018	27.3	0		
	140	141 161	AA2 AB2	21RA107AA* 21RA107AB*													
	80	81	XS3	31RA107AB													
	104	105	XA3	31RA107A*	100	05	10117	+0.021		07.5	_	6	.0.015	21.8	+0.2		
	120	121	XB3	31RA107B*	120	35	19H7	0	-	37.5	- 1	6	±0.015	21.8	0		
	152	153	XC3	31RA107C*													
	80	81	XD3	31RA107D*				. 0 000									
	104	105 121	XE3 XF3	31RA107E* 31RA107F*	99	35	14	+0.030	-	21.5	-	5	±0.015	16.3	+0.2		
	152	153	XG3	31RA107G*				+0.012							"		
	80	81	XH3	31RA107H*													
RA-40E	104	105	XJ3	31RA107J*	150	42	24H7	+0.021	_	47.5	_	8	±0.018	27.3	+0.2		
TIA-40L	120	121	XK3	31RA107K*	130	42	24111	0	_	47.5	_	0	±0.010	21.5	0		
	152	153	XL3	31RA107L*								-		<u> </u>			
	80 104	81 105	XM3 XN3	31RA107M* 31RA107N*			+0.036							+0.2			
	120	121	XP3	31RA107P*	142.5	42	2 24	+0.036		45	-	8	±0.018	27.3	0		
	152	153	XQ3	31RA107Q*				10.010									
	80	81	XR3	31RA107R*													
	104	105	XT3	31RA107T*	150	50	32	32	32	+0.043	_	50.5	_	10	±0.018	35.3	+0.2
	120	121	XU3	31RA107U*			"-	+0.018		00.0			20.0.0	00.0	0		
	152 80	153 81	XW3 XS4	31RA107W* 41RA107-*													
	100	101	XA4	41RA107A*				+0.021							+0.2		
	120	121	XB4	41RA107B*	96	45	19H7	0	-	37.5	-	6	±0.015	21.8	0		
	152	153	XC4	41RA107C*													
	80	81	BD4	41RA107BD*													
	100	101	BE4 BF4	41RA107BE* 41RA107BF*	177	55	35	+0.035	-	72.5	-	10	±0.018	38.3	+0.2		
	152	153	BG4	41RA107BG*				+0.010							0		
	80	81	XH4	41RA107BG													
DA 005	100	101	XJ4	41RA107J*	105	45	04117	+0.021		14 =		C	.0.010	07.0	+0.2		
RA-80E	120	121	XK4	41RA107K*	125	45	24H7	0	-	41.5	-	8	±0.018	27.3	0		
	152	153	XL4	41RA107L*													
	100	81 101	XM4 XN4	41RA107M* 41RA107N*				+0.036									
	120	121	XN4 XP4	41RA107N** 41RA107P*	128.5	45	24	+0.036	-	45	-	8	±0.018	27.3	+0.2		
	152	153	XQ4	41RA107Q*				10.010							"		
	80	81	XR4	41RA107R*													
	100	101	XT4	41RA107T*	156	50	32	+0.043	_	50.5	_	10	±0.018	35.3	+0.2		
	120	121	XU4	41RA107U*	100	30	52	+0.018	-	00.0		10	10.010	00.0	0		
	152	153	XW4	41RA107W* 51RA107-*						-							
	100	81 101	XS5 XA5	51RA107-^ 51RA107A*													
	128	129	XB5	51RA107A 51RA107B*	205	58	35	+0.035	_	72	_	10	±0.018	38.3	+0.2		
	144	145	XC5	51RA107C*		50	50	+0.010		'-			=0.010	55.5	0		
	170	171	XD5	51RA107D*													
	80	81	XE5	51RA107E*													
DA 4005	100	101	XF5	51RA107F*	4.40		0.41.17	+0.021		1 ,			0010	07.0	+0.2		
RA-160E	128	129	XG5	51RA107G*	140	50	24H7	0	-	43	-	8	±0.018	27.3	0		
	144	145 171	XH5 XJ5	51RA107H* 51RA107J*													
	80	81	XK5	51RA1073 51RA107K*													
	100	101	XL5	51RA107L*				.0.040									
	128	129	XM5	51RA107M*	164	58	32	+0.043 +0.018	-	50	-	10	±0.018	35.3	+0.2		
		145	XN5	51RA107N*		I	1	+0.010	1	1	1		1	I	1		
	144 170	171	XP5	51RA107P*											1		

Straight shaft (without key)

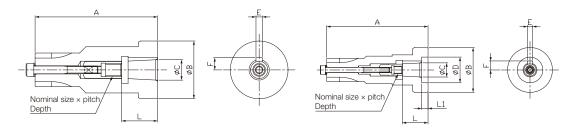


Model		d ratio	Input gear code	Order item number				sions (mm)		Transmission to	
Model	EA	EC	Tilput gear code	Order iterritianiber	А	ØB	9	2C	L	Nm	
	80	81	AC2	21RA405AC*							
	104	105	AD2	21RA405AD*				0.000			
	120	121	AE2	21RA405AE*	113	63	19	+0.036	35	52.4	
	140	141	AF2	21RA405AF*				+0.015			
	160	161	AG2	21RA405AG*							
	80	81	AH2	21RA405AH*							
	104	105	AJ2	21RA405AJ*							
RA-20E	120	121	AK2	21RA405AK*	123	63	24	+0.036	45	77.8	
	140	141	AL2	21RA405AL*			-	+0.015			
	160	161	AM2	21RA405AM*							
	80	81	AN2	21RA405AN*							
	104	105	AP2	21RA405AP*							
	120	121	AQ2	21RA405AQ*	120.5	63	24H7	+0.021	47.5	77.8	
	140	141	AR2	21RA405AR*				0			
	160	161	AT2	21RA405AT*							
	80	81	AZ3	31RA405AZ*							
	104	105	BA3	31RA405BA*				+0.036			
	120	121	BB3	31RA405BB*	132.5	63	24	+0.015	45	77.8	
	152	153	BC3	31RA405BC*				10.0.0			
	80	81	AB3	31RA405AB*							
	104	105	AC3	31RA405AC*				+0.043			
	120	121	AD3	31RA405AD*	150	68	32	+0.018	50.5	170.8	
	152	153	AE3	31RA405AE*				10.010			
RA-40E	80	81	AF3	31RA405AF*							
	104	105	AG3	31RA405AG*				+0.021			
	120	121	AH3	31RA405AH*	120	63	19H7	0	37.5	52.4	
	152	153	AJ3	31RA405AJ*							
	80	81	AK3	31RA405AK*							
	104	105	AL3	31RA405AL*				+0.021			
	120	121	AM3	31RA405AM*	127.5	63	24H7	0	45	77.8	
	152	153	AN3	31RA405AN*							
	80	81	XX4	41RA405X*							
	100	101	XY4	41RA405Y*				+0.036			
	120	121	XZ4	41RA405Z*	128.5	68	24	+0.036	45	77.8	
	152	153	AA4	41RA405AA*				+0.015			
	80	81	AB4	41RA405AB*							
	100	101	AC4	41RA405AC*				+0.043			
	120	121	AD4	41RA405AC*	136	68	32	+0.043	50.5	170.8	
	152	153	AE4	41RA405AE*				+0.010			
	80	81	AF4	41RA405AE*							
	100	101	AG4	41RA405AG*				+0.021			
RA-80E	120	121	AH4	41RA405AG*	131	68	19H7	0	37.5	52.4	
	152	153	AJ4	41RA405AH 41RA405AJ*				0			
	80	81	AK4	41RA405AK*							
	100	101	AL4	41RA405AL*				+0.035			
	120	121	AM4	41RA405AM*	157	68	35		72.5	106.5	
	152	153	AN4					+0.010			
			AN4 AP4	41RA405AN*			-		-	-	
	80	81		41RA405AP*				+0.021			
	100	101	AQ4	41RA405AQ*	135	68	24H7	+0.021	41.5	77.8	
	120	121	AR4 AT4	41RA405AR*				0			
	152 80			41RA405AT*			-			-	
	100	81 101	XQ5 XR5	51RA405Q* 51RA405R*							
	128	129			164	0.4	32	+0.043	E0	170.0	
			XT5	51RA405T*	164	84	32	+0.018	50	170.8	
	144	145	XU5	51RA405U*							
	170	171	XW5	51RA405W*			-				
	80	81	XX5	51RA405X*							
DA 400E	100	101	XY5	51RA405Y*	400	0.4	0.5	+0.035	07	054.7	
RA-160E	128	129	XZ5	51RA405Z*	180	84	35	+0.010	67	251.7	
	144	145	AA5	51RA405AA*				10.0.0			
	170	171	AB5	51RA405AB*							
	80	81	AC5	51RA405AC*							
	100	101	AD5	51RA405AD*				+0.021	I I		
	128	129	AE5	51RA405AE*	160	84	24H7	0	43	77.8	
	144	145	AF5	51RA405AF*				'			
	170	171	AG5	51RA405AG*		1	1				

The asterisk (*) at the end of the order item number will be replaced with our revision number. It is subject to change without notice.

The drawing above shows a representative model. The actual model may differ from the drawing.

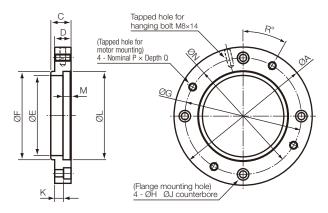
1/10 tapered shaft



Model	Speed	d ratio	Input gear	Order item number				lr	nput gea	ır dimer	isions (n	nm)				Thre	aded por	tion
iviodei	EA	EC	code	Order item number	Α	ØB	Q	OC OC	ØD	L	L1		E	F	F	Nominal size	Pitch	Depth
	80	81	BA2	21RA406BA*														
	104	105	BB2	21RA406BB*	1 1			+0.1					+0.040			İ		
	120	121	BC2	21RA406BC*	79	35	11	+0.1	20	20	5	4	+0.040	7	+0.2	M6	P1.0	8.5
	140	141	BD2	21RA406BD*	1 1			0					+0.010		0			
BA-20F	160	161	BE2	21RA406BE*	1 1													
NA-ZUE	80	81	BF2	21RA406BF*														
	104	105	BG2	21RA406BG*]			+0.1					+0.040		+0.2			
	120	121	BH2	21RA406BH*	89	35	14	0.1	-	18	-	4	+0.040	8.5	1 +0.2	M8	P1.0	14
	140	141	BJ2	21RA406BJ*]			0					+0.010		"			
	160	161	BK2	21RA406BK*														
	80	81	AP3	31RA406AP*														
	104	105	AQ3	31RA406AQ*	119.5	35	16	+0.1		28	_	5	+0.040	9.5	+0.2	M10	P1.25	16
	120	121	AR3	31RA406AR*	119.5	55	10	0	_	20	_	3	+0.010	9.0	0	IVITO	F 1.20	10
RA-40E	152	153	AT3	31RA406AT*														
NA-40E	80	81	AU3	31RA406AU*														
	104	105	AW3	31RA406AW*	93.5	35	14	+0.1	_	18	_	4	+0.040	8.5	+0.2	M8	P1.0	14
	120	121	AX3	31RA406AX*	30.0	33	14	0	_	10	_	4	+0.010	0.5	0	IVIO	11.0	14
	152	153	AY3	31RA406AY*														
	80	81	AU4	41RA406AU*]													
	100	101	AW4	41RA406AW*	95.5	45	16	+0.1		28	_	5	+0.040	9.5	+0.2	M10	P1.25	16
	120	121	AX4	41RA406AX*	35.5	40	10	0	_	20	_	3	+0.010	9.0	0	IVITO	11.20	10
RA-80E	152	153	AY4	41RA406AY*														
TIA OOL	80	81	AZ4	41RA406AZ*]													
	100	101	BA4	41RA406BA*	141.5	50	32	+0.1	_	58	_	7	+0.080	17.75	+0.2	M20	P1.5	23
	120	121	BB4	41RA406BB*	141.0	00	02	0		00		,	+0.043	17.70	0	IVIZO	1 1.0	20
	152	153	BC4	41RA406BC*														
	80	81	AH5	51RA406AH*														
	100	101	AJ5	51RA406AJ*				+0.1					+0.080		+0.2			
	128	129	AK5	51RA406AK*	170	50	32	0	-	58	-	7	+0.043	17.75	0	M20	P1.5	23
	144	145	AL5	51RA406AL*				~					1.0.0-70		"			
BA-160F	170	171	AM5	51RA406AM*	\square								1					
TOOL	80	81	AN5	51RA406AN*														
	100	101	AP5	51RA406AP*				+0.1					+0.040		+0.2			
	128	129	AQ5	51RA406AQ*	134	50	16	0	-	28	-	5	+0.010	9.5	0	M10	P1.25	16
	144	145	AR5	51RA406AR*				"					10.010					
	170	171	AT5	51RA406AT*														

External dimensions Motor flange

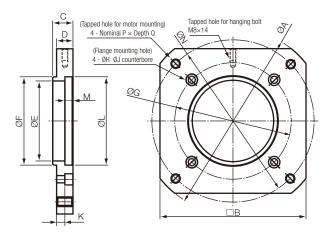
Round type



Model	Motor flange	Order item	Flange	outer di	mension	s (mm)	Reduc	ction gear	mount	ing dime	ensions	(mm)		Motor	mounti	ng dime	nsions (mm)	
iviodei	code	number	ØA	С	D	ØE	Q)F	ØG	ØH	ØJ	K	Q)L	М	ØN	Р	Q	R°
RA-20E	2XB	21RA203B*	123	19	15	70	80h7	0 -0.030	100	6.6	11	7	80H7	+0.030	6	100	M6	15	45
TIA-ZUL	2XD	21RA203D*	123	34	30	70	80h7	0 -0.030	100	6.6	11	7	80	+0.037 +0.012	10	100	M6	15	45
	4XR	20S203R*	170	30	25	100	110h7	0 -0.035	145	9	14	10	110H7	+0.035	10	145	M8	20	30
	4XT	20S203T*	170	25	20	100	110h7	0 -0.035	145	9	14	10	110	+0.038 +0.013	10	130	M8	20	30
RA-80E	4XU	20S203U*	170	40	35	100	110h7	0 -0.035	145	9	14	10	110H7	+0.035	10	145	M8	20	30
	4AA	20S203AA*	190	55	50	100	110h7	0 -0.035	150	9	14	10	130	+0.039 +0.014	10	165	M10	20	45
	4AB	20S203AB*	190	35	30	100	110h7	0 -0.035	150	9	14	10	130	+0.039 +0.014	10	165	M10	20	45
RA-160E	5XS	20\$203-*	170	25	20	100	110h7	0 -0.035	145	9	14	10	110H7	+0.035	10	145	M8	20	30
TIATOOL	5XY	20S203Y*	190	25	20	100	110h7	0 -0.035	150	9	14	10	130	+0.039 +0.014	10	165	M10	20	45

The asterisk (*) at the end of the order item number will be replaced with our revision number. It is subject to change without notice. The drawing above shows a representative model. The actual model may differ from the drawing.

Square type



Model	Motor flange	Order item	Flar	nge oute	er dimer	sions (n	nm)	Reduc	ction gear	mount	ing dime	ensions	(mm)	M	otor mou	nting di	mensior	ns (mm)	
iviodei	code	number	ØA	□B	С	D	ØE	Q	F	ØG	ØH	ØJ	K	Ø	L	М	ØN	Р	Q
	2XC	21RA203C*	165	130	59	55	70	80h7	0 -0.030	95	6.6	11	7	110H7	+0.035 0	10	145	M8	20
RA-20E	2XE	21RA203E*	165	130	59	55	70	80h7	0 -0.030	95	6.6	11	7	110	+0.038 +0.013	10	130	M8	20
NA-ZUE	2XF	21RA203F*	165	130	44	40	70	80h7	0 -0.030	95	6.6	11	7	110	+0.038 +0.013	10	130	M8	20
	2XG	21RA203G*	165	130	44	40	70	80h7	0 -0.030	95	6.6	11	7	110H7	+0.035	10	145	M8	20
	3XA	10S203A*	165	130	35	30	72	80h7	0 -0.030	95	9	14	10	110H7	+0.035	11	145	M8	20
	3XF	10S203F*	165	130	55	50	72	80h7	0 -0.030	95	9	14	10	110H7	+0.035	11	145	M8	20
RA-40E	3AG	10S203AG*	165	130	45	40	72	80h7	0 -0.030	95	9	14	10	110	+0.038 +0.013	10	130	M8	20
	ЗАН	10S203AH*	190	150	55	50	72	80h7	0 -0.030	95	9	14	10	130	0.039 +0.014	12	165	M10	20
	3AJ	10S203AJ*	165	130	35	30	72	80h7	0 -0.030	95	9	14	10	110	+0.038 +0.013	10	130	M8	20
RA-80E	4XQ	20S203Q*	230	180	75	70	100	110h7	0 -0.035	145	9	14	10	114.3H7	+0.035	10	200	M12	20
nA-ouE	4AC	20S203AC*	230	180	55	50	100	110h7	0 -0.035	145	9	14	10	114.3H7	+0.035	10	200	M12	20
RA-160E	5XD	20S203D*	230	180	65	60	100	110h7	0 -0.035	145	9	14	10	114.3H7	+0.035 0	10	200	M12	20
11A-100E	5XZ	20S203Z*	230	180	45	40	100	110h7	0 -0.035	145	9	14	10	114.3H7	+0.035	10	200	M12	20

The asterisk (*) at the end of the order item number will be replaced with our revision number. It is subject to change without notice. The drawing above shows a representative model. The actual model may differ from the drawing.



Considering the use of the RA Series

This product features high precision and high rigidity, however, it is necessary to strictly comply with various restrictions and make appropriate to maximize the product's features. Please read this technical document thoroughly and select and adopt an appropriate model based on the actual operating environment, method, and conditions at your facility.

Export

When this product is exported from Japan, it may be subject to the export regulations provided in the "Foreign
Exchange Order and Export Trade Control Order". Be sure to take sufficient precautions and perform the required
export procedures in advance if the final operating party is related to the military or the product is to be used in the
manufacture of weapons, etc.

Application

• If failure or malfunction of the product may directly endanger human life or if it is used in units which may injure the human body (atomic facilities, space equipment, medical equipment, safety units, etc.), examination of individual situations is required. Contact our agent or nearest business office in such a case.

Safety measures

Although this product has been manufactured under strict quality control, a mistake in operation or misuse can
result in breakdown or damage, or an accident resulting in injury or death. Be sure to take all appropriate safety
measures, such as the installation of independent safeguards.

Product specifications indicated in this catalog

• The specifications indicated in this catalog are based on Nabtesco evaluation methods. This product should only be used after confirming that it is appropriate for the operating conditions of your system.

Operating environment

Use the reduction gear under the following environment:

- · Location where the ambient temperature is within the range from -10°C to 40°C.
- · Location where the humidity is less than 85% and no condensation occurs.
- · Location where the altitude is less than 1000 m.
- · Well-ventilated location

Do not install the reduction gear at the following locations.

- · Locations where a lot of dust is collected.
- \cdot Outdoor areas that are directly affected by wind and rain
- · Locations near to areas that contains combustible, explosive, or corrosive gases and flammable materials.
- · Locations that are heated due to heat transfer and radiation from peripherals and direct sun.
- \cdot Locations where the performance of the motor can be affected by magnetic fields or vibration.

Note 1: If the required operating environment cannot be established/met, contact us in advance.

2: When using the reduction gear under special conditions (clean room, equipment for food, concentrated alkali, high-pressure steam, etc.), contact our agent or nearest business office in advance.

Maintenance

• The standard replacement time for lubricant is 20,000 hours. However, when operation involves a reduction gear surface temperature above 40°C, the state of degradation of the lubricant should be checked in advance of that and the grease replaced earlier as necessary.

Reduction gear temperature

• When the reduction gear is used under high load and at a high duty ratio, it may overheat and the surface temperature may exceed the allowable temperature. Be aware of conditions so that the surface temperature of the reduction gear does not exceed 60°C while it is in operation. There is a possibility of damage (to the product) if the surface temperature exceeds 60°C.

Reduction gear output rotation angle

 When the range of the rotation angle is small (10 degrees or less), the service life of the reduction gear may be reduced due to poor lubrication or the internal parts being subject to a concentrated load.

Note: Contact us in case the rotation angle is 10 degrees or less.

Manuals

• Safety information and detail product instructions are indicated in the operation manual. The operation manual can be downloaded from the following website.

http://precision.nabtesco.com/

Glossary

Rated service life

The lifetime resulting from the operation with the rated torque and the rated output speed is referred to as the "rated service life".

Allowable acceleration/deceleration torque

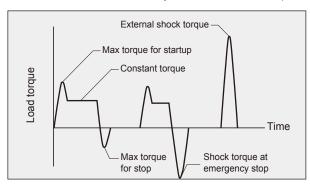
When the machine starts or stops, the load torque to be applied to the reduction gear is larger than the constant-speed load torque due to the effect of the inertia torque of the rotating part. In such a situation, the allowable torque during acceleration/deceleration is referred to as "allowable acceleration/deceleration torque".

Note: Be careful that the load torque, which is applied at startup and stop, does not exceed the allowable acceleration/deceleration torque.

Momentary maximum allowable torque

A large torque may be applied to the reduction gear due to execution of emergency stop or by an external shock. In such a situation, the allowable value of the momentary applied torque is referred to as "momentary maximum allowable torque".

Note: Be careful that the momentary excessive torque does not exceed the momentary maximum allowable torque.



Allowable output speed

The allowable value for the reduction gear's output speed during operation without a load is referred to as the "allowable output speed".

Notes: Depending on the conditions of use (duty ratio, load, ambient temperature), the reduction gear temperature may exceed 60°C even when the speed is under the allowable output speed. In such a case, either take cooling measures or use the reduction gear at a speed that keeps the surface temperature at 60°C or lower.

Duty ratio

The duty ratio is defined as the ratio of the sum total time of acceleration, constant speed, and deceleration to the cycle time of the reduction gear.

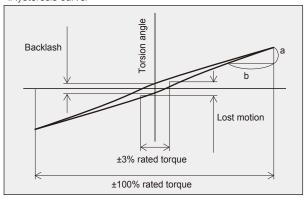
Torsional rigidity, lost motion, backlash

When a torque is applied to the output shaft while the input shaft is fixed, torsion is generated according to the torque value. The torsion can be shown in the hysteresis curves.

The value of b/a is referred to as "torsional rigidity".

The torsion angle at the mid point of the hysteresis curve width within ±3% of the rated torque is referred to as "lost motion". The torsion angle when the torque indicated by the hysteresis curve is equal to zero is referred to as "backlash".

<Hysteresis curve>



Startup efficiency

The efficiency of the moment when the reduction gear starts up is referred to as "startup efficiency".

No-load running torque (input shaft)

The torque for the input shaft that is required to run the reduction gear without load is referred to as "no-load running torque".

Allowable moment and maximum thrust load

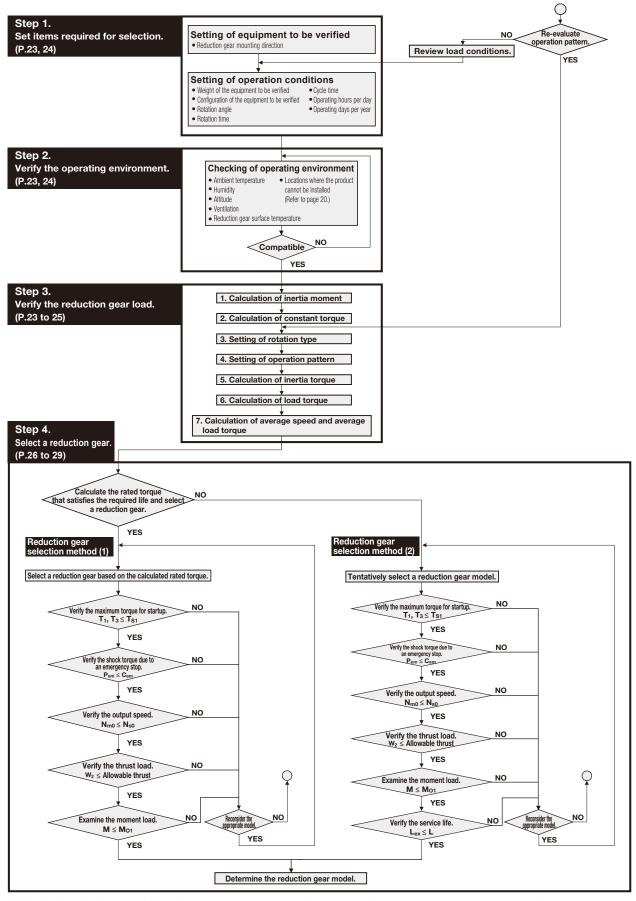
The external load moment may be applied to the reduction gear during normal operation. The allowable values of the external moment and the external axial load at this time are each referred to as "allowable moment" and "maximum thrust load".

Momentary maximum allowable moment

A large moment may be applied to the reduction gear due to an emergency stop or external shock. The allowable value of the momentary applied moment at this time is referred to as "momentary maximum allowable moment."

Note: Be careful so that the momentary excessive moment does not exceed the momentary maximum allowable moment.

Product selection flowchart



A limitation is imposed on the motor torque value according to the momentary maximum allowable torque of the selected reduction gear. (Refer to page 30.)

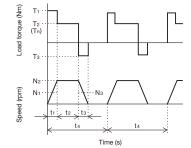
Product selection

Model code selection examples

With horizontal rotational transfer

Step 1. Set the items required for selection.

Setting item	Setting
Reduction gear mounting direction	Vertical shaft installation
Equipment weight to be considered	
W _A Disk weight (kg)	180
W _B — Work weight (kg)	20×4 pieces
Equipment configuration to be considered	
D ₁ Disk: D dimension (mm)	1,200
a — Workpiece: a dimension (mm)	100
b — Workpiece: b dimension (mm)	300
D ₂ — Workpiece: P.C.D. (mm)	1,000
Operation conditions	
θ ————Rotation angle (°)*1	180
$[t_1+t_2+t_3]$ —— Rotation time (s)	2.5
[t ₄] ———— Cycle time (s)	20
Q ₁	12
Q2Equipment operation days per year (days/year)	365



^{*1.} When the range of the rotation angle is small (10 degrees or less), the rating life of the reduction gear may be reduced due to poor lubrication or the internal parts being subject to a concentrated load.

Step 2. Verify the operating environment.

Checkpoint	Standard value
S ₀ ———— Ambient temperature (°C)	-10 to +40
S ₁ ———— Reduction gear surface temperature (°C)	60 or less

Note: Refer to "Operating environment" on p. 20 for values other than those listed above.

60 40 -10 -10 S₀(°C)

Step 3-1. Examine the reduction gear load.

	Setting item	Calculation formula	Selection examples
(1) Calculate the	inertia moment based the calculat	ion formula on page 36.	
I _R	Load inertia moment (kgm²)	$\begin{split} I_{R1} &= \frac{W_A \times \left(\frac{D_1}{2 \times 1,000}\right)^2}{2} \\ I_{R2} &= \left[\frac{W_B}{12} \left\{\left(\frac{a}{1,000}\right)^2 + \left(\frac{b}{1,000}\right)^2\right\} + W_B \times \left(\frac{D_2}{2 \times 1,000}\right)^2\right] \times n \\ I_{R1} &= \text{Disk inertia moment} \\ I_{R2} &= \text{Work inertia} \\ I_{R} &= I_{R1} + I_{R2} \\ n &= \text{Number of Workpieces} \end{split}$	$\begin{split} I_{R1} &= \frac{180 \times \left(\frac{1,200}{2 \times 1,000}\right)^2}{2} \\ &= 32.4 \text{ (kgm}^2) \\ I_{R2} &= \left[\frac{20}{12} \left\{ \left(\frac{100}{1,000}\right)^2 + \left(\frac{300}{1,000}\right)^2 \right] + 20 \times \left(\frac{1,000}{2 \times 1,000}\right)^2 \right] \times 4 \\ &= 20.7 \text{ (kgm}^2) \\ I_{R} &= 32.4 + 20.7 \\ &= 53.1 \text{ (kgm}^2) \end{split}$
(2) Examine the c	constant torque.		
T _R	Constant torque (Nm)	$\begin{split} &T_{\text{R}}\!=\!(W_{\text{A}}\!+\!W_{\text{B}})\!\times 9.8\times\frac{D_{\text{in}}}{2\!\times\!1,000}\!\times\!\mu\\ &\mu\!=\!\text{Friction factor} \\ &\text{Note: Use 0.015 for this example as the load}\\ &\text{is applied to the bearing of the}\\ &\text{precision reduction gear.} \\ &D_{\text{in}}\!=\!\text{Rolling diameter: Use the pilot diameter}\\ &\text{which is almost equivalent}\\ &\text{to the rolling diameter in}\\ &\text{this selection calculation.} \\ &\text{Note: If the reduction gear model is not determined,}\\ &\text{select the following pilot diameter:}\\ &\text{Maximum pilot diameter: 240 (mm)}\\ &\text{(RA-160EA)} \end{split}$	$T_R = (180 + 20 \times 4) \times 9.8 \times \frac{240}{2 \times 1,000} \times 0.015$ = 4.6 (Nm)

Step 3-2: Proceed to p. 25.

Product selection

Model code selection examples

Motor flange

Rotation center

Load torque (Nm)

Rotation speed (rpm)

60

40

Ö

With vertical rotational transfer

Step 1. Set the items required for selection.

Setting item	Setting
Reduction gear mounting direction	Horizontal shaft installation
Equipment weight to be considered	
W _C ——— Mounted work weight (kg)	490
Equipment configuration to be considered	
a ———— a dimension (mm)	500
b ———— b dimension (mm)	500
R R dimension (mm)	320
Operation conditions	
θ ———— Rotation angle (°)*1	90
$[t_1+t_2+t_3]$ — Rotation time (s)	1.5
[t ₄] ———— Cycle time (s)	20
Q ₁ ———— Equipment operation hours per day (hours/day)	24
Q ₂ ——— Equipment operation days per year (days/year)	365

^{*1.} When the range of the rotation angle is small (10 degrees or less), the rating life of the reduction gear may be reduced due to poor lubrication or the internal parts being subject to a concentrated load.

Step 2. Verify the operating environment.

Checkpoint	Standard value
S ₀ — Ambient temperature (°C)	-10 to +40
S ₁ ———— Reduction gear surface temperature (°C)	60 or less

S ₁ ——— F	Reduction gear surface temperatur	re (°C)	60 or less	ي. 0-ره	
Note: Refer to "C	Note: Refer to "Operating environment" on p. 20 for values other than those listed above.			-10	
Step 3-1. Examine the reduction gear load.				-10 40 S ₀ (°C)	
	Setting item		Calculation formula		Selection examples
(1) Calculate th	e inertia moment.				
I _R	Load inertia moment (kgm²)	$I_{R} = \frac{W_{C}}{12} \times \left\{ \left(\frac{a}{1.00} \right) \right\}$	$ \sqrt{\frac{b}{1,000}} + \left(\frac{b}{1,000}\right)^2 + W_C \times \left(\frac{R}{1,000}\right)^2 $	$I_{R} = \frac{490}{12} \times \left\{ \left(\frac{500}{100} \right) \right\}$ $= 70.6 \text{ (kgm²)}$	$\left(\frac{500}{0}\right)^{2} + \left(\frac{500}{1,000}\right)^{2} + 490 \times \left(\frac{320}{1,000}\right)^{2}$
(2) Examine the	constant torque.				
T _R	Constant torque (Nm)	$T_{R} = W_{C} \times 9.8$	< R 1,000	T _R = 490 × 9.8 × = 1,537 (Nm)	320 1,000

Step 3-2: Proceed to p. 25.

(Refer to "With horizonta5rotational transfer" for selection examples.)

Step 3-2. Set items required for selection

	Setting item	Calculation formula	Selection examples (With horizontal rotational transfer)
(3) Set the r	rotation type.		
	ear rotation type		Case rotation type (RA-EA series)
(4) Set the a	acceleration/deceleration time	, constant-speed operation time, and output spee	 Examine the operation pattern using N₂ = 15 rpm as the
t ₁	— Acceleration time (s)	The operation pattern does not need to be verified if it is already set. If the operation pattern has not been determined, use the following formula to calculate the reference operation pattern.	reduction gear output speed is unknown.
t ₂	Constant-speed operation time (s)	$t_1 = t_3 = \text{Rotation } [t_1 + t_2 + t_3] - \frac{\theta}{\left(\frac{N_2}{60} \times 360\right)}$	$t_1 = t_3 = 2.5 - \frac{180}{\left(\frac{15}{60} \times 360\right)} = 0.5 \text{ (s)}$ $t_2 = 2.5 - (0.5 + 0.5) = 1.5 \text{ (s)}$
t ₃	Deceleration time (s)	t_2 = Rotation [$t_1+t_2+t_3$] – (t_1+t_3) Note: 1. Assume that t_1 and t_3 are the same.	$\therefore t_1 = t_3 = 0.5 \text{ (s)}$
N ₂	Constant speed (rpm)	Note: 2. $N_2 = 15$ pm if the reduction gear output speed (N_2) is not known. Note: 3. If t_1 and t_3 is less than 0, increase the output speed or extend the rotation time.	t ₂ = 1.5 (s) N ₂ = 15 (rpm)
N ₁	Average speed for startup (rpm)	$N_1 = \frac{N_2}{2}$	$N_1 = \frac{15}{2} = 7.5 \text{ (rpm)}$
N ₃	Average speed for stop (rpm)	$N_3 = \frac{N_2}{2}$	$N_3 = \frac{15}{2} = 7.5 \text{ (rpm)}$
(5) Calculat	e the inertia torque for acceler	ration/deceleration.	
Та	Inertia torque for acceleration (Nm)	$T_A = \left\{ \frac{I_R \times (N_2 - 0)}{t_1} \right\} \times \frac{2\pi}{60}$	$T_{A} = \left\{ \frac{53.1 \times (15 - 0)}{0.5} \right\} \times \frac{2\pi}{60}$ = 166.8 (Nm)
T _D	Inertia torque for deceleration (Nm)	$T_{D} = \left\{ \frac{I_{R} \times (0 - N_{2})}{t_{3}} \right\} \times \frac{2\pi}{60}$	$T_{D} = \left\{ \frac{53.1 \times (0 - 15)}{0.5} \right\} \times \frac{2\pi}{60}$ $= -166.8 \text{ (Nm)}$
(6) Calculat	e the load torque for accelerat	ion/deceleration.	
T ₁	Maximum torque for startup (Nm)	$\begin{split} &T_3 = \left T_A + T_R\right \\ &T_R : \text{Constant torque} \\ & \underline{\text{With horizontal rotational transfer}} \text{Refer to page 23} \\ & \underline{\text{With vertical rotational transfer}} \text{Refer to page 24} \end{split}$	$T_1 = 166.8 + 4.6 $ = 171.4 (Nm)
T ₂	Constant maximum torque (Nm)	$T_2 = T_R $	T ₂ =4.6 (Nm)
T ₃	Maximum torque for stop (Nm)	$\begin{split} T_3 = & \left T_A + T_R \right \\ T_R : \text{Constant torque} \\ & \underline{\text{With horizontal rotational transfer}} \text{Refer to page 23} \\ & \underline{\text{With vertical rotational transfer}} \text{Refer to page 24} \end{split}$	T ₃ = -166.8+4.6 =162.2 (Nm)
(7)-1 Calcul	ate the average speed.		
	— Average speed (rpm)	$N_{m} = \frac{t_{1} \times N_{1} + t_{2} \times N_{2} + t_{3} \times N_{3}}{t_{1} + t_{2} + t_{3}}$	$N_{m} = \frac{0.5 \times 7.5 + 1.5 \times 15 + 0.5 \times 7.5}{0.5 + 1.5 + 0.5}$ $= 12 \text{ (rpm)}$
(7)-2 Calcul	ate the average load torque.		
T _m	— Average load torque (Nm)	$T_{m} = \sqrt[10]{\frac{10}{3} t_{1} \times N_{1} \times T_{1}^{\frac{10}{3}} + t_{2} \times N_{2} \times T_{2}^{\frac{10}{3}} + t_{3} \times N_{3} \times T_{3}^{\frac{10}{3}}}{t_{1} \times N_{1} + t_{2} \times N_{2} + t_{3} \times N_{3}}}$	$T_{m} = \sqrt[3]{0.5 \times 7.5 \times 171.4^{\frac{10}{3}} + 1.5 \times 15 \times 4.6^{\frac{10}{3}} + 0.5 \times 7.5 \times 162.2^{\frac{10}{3}}} \\ = 110.1 \text{ (Nm)}$

Go to page 26 if the reduction gear model is verified based on the required life. Go to page 28 if the service life is verified based on the reduction gear model.

Product selection

Model code selection examples

Step 4. Select a reduction gear

Reduction gear selection method (1) Calculate the required torque based on the load conditions and required life and select a reduction gear.

Setting/verific	ation item	Calculation formula	Selection examples (With horizontal rotational transfer)	
(1) Calculate the rated torque for the reduction gear that satisfies the required life.				
Lex Required	life (year)	Based on the operation conditions	5 years	
Q _{1cy} ———— Number of cycles per day (times)		$Q_{tcy} = \frac{Q_1 \times 60 \times 60}{t_4}$	$Q_{toy} = \frac{12 \times 60 \times 60}{20}$ = 2,160 (times)	
Q ₃ — Operating gear per o	g hours of reduction day (h)	$Q_3 = \frac{Q_{loy} \times (t_1 + t_2 + t_3)}{60 \times 60}$	$Q_3 = \frac{2,160 \times (0.5 + 1.5 + 0.5)}{60 \times 60}$ = 1.5 (h)	
Q ₄ — Operating gear per y	g hours of reduction year (h)	$Q_4 = Q_3 \times Q_2$	Q ₄ =1.5×365 =548 (h)	
L _{hour} — Reduction	n gear service life (h)	$Lhour = Q_4 \times L_ex$	Lhour = 548 × 5 = 2,740 (h)	
	n gear rated torque fies the required life	$T_0' = T_m \times \left(\frac{10}{3}\right) \sqrt{\frac{L_{hour}}{K} \times \frac{N_m}{N_0}}$ $K : \text{Reduction gear rated life (h)}$ $N_0 : \text{Reduction gear rated output speed (rpm)}$	To' = 110.1× $\frac{\binom{10}{3}}{\binom{2.740}{6,000}}$ × $\frac{12}{15}$ = 81.5(Nm)	
(2) Tentatively select a	reduction gear mod	lel based on the calculated rated torque.		
Tentative selection of the reduction gear		Select a reduction gear for which the rated torque of the reduction gear [T ₀] ^T is equal to or greater than the rated torque of the reduction gear that satisfies the required life [T ₀]. *1 [T ₀]: Refer to the rating table on page 8	RA-20EA that meets the following condition is tentatively selected: [To] 167 (Nm) \geq [To'] 81.5 (Nm)	
(3) Verify the maximum	torque for startup	and stop.		
Verification of maximum torque for startup and stop		Check the following conditions: The allowable acceleration/deceleration torque $[T_{51}]^{11}$ is equal to or greater than the maximum starting torque $[T_{1}]^{2}$ and maximum stopping torque $[T_{2}]^{2}$ If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model.	$ [T_{s1}] \ 412 \ (Nm) \geq [T_1] \ 171.4 \ (Nm) $ $ [T_S] \ 162.2 \ (Nm) $ According to the above conditions, the tentatively selected model should be no problem.	
		*2 $[T_1]$ and $[T_3]$: Refer to page 24		
(4) Verify the output sp	eed.			
N _{m0} Average speed per cycle (rpm)		$N_{\text{m0}} = \frac{t_1 \times N_1 + t_2 \times N_2 + t_3 \times N_3}{t_4}$	$N_{m0} = \frac{0.5 \times 7.5 + 1.5 \times 15 + 0.5 \times 7.5}{20}$ $= 1.5 \text{ (rpm)}$	
Verification of output speed		Check the following condition: The allowable output speed (100% duty ratio) [N _{s0}] ⁻¹ is equal to or greater than the average speed per cycle [N _{m0}] If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model. Contact us regarding use of the model at a speed outside the allowable output speed (40% duty ratio) [N _{S1}] ⁻¹ . Note: The value of [N _{S0}] is the speed at which the case temperature is balanced at 60°C for 30 minutes. *1 [N _{S0}] and [N _{S1}]: Refer to the rating table on page 9	$[N_{s0}]~45~(rpm)\geq [N_{m0}]~1.5~(rpm)$ According to the above condition, the tentatively selected model should be no problem.	

Reduction gear selection method (1) Calculate the required torque based on the load conditions and required life and select a reduction gear.

Verify the shock torque at the time of Expected number of emergency stop times (time Shock torque due to an emergency stop (Nm) Speed at the time of an emergency stop (rpm) Deceleration time at the time of an emergency stop (s)	Based on the operation conditions. (i) I) II II II II II II II II	For example, an emergency stop occurs once a month. $[P_{em}] = 1 \times 12 \times \text{required life (year) } [L_{ex}] \\ = 12 \times 5 = 60 \text{ (times)}$ For example, $[T_{em}] = 500 \text{ (Nm)}$ For example, $[N_{em}] = 15 \text{ (rpm)}$
shock torque due to an emergency stop (Nm) Speed at the time of an emergency stop (rpm) Deceleration time at the time	(md) poeds (man) Time (s)	$[P_{em}] = 1 \times 12 \text{ x required life (year) } [L_{ex}]$ $= 12 \times 5 = 60 \text{ (times)}$ $\text{For example, } [T_{em}] = 500 \text{ (Nm)}$ $\text{For example, } [N_{em}] = 15 \text{ (rpm)}$
emergency stop (Nm) Speed at the time of an emergency stop (rpm) Deceleration time at the time	N _{on} N _{on} N _{on} Time (s)	For example, [N _{em}] = 15 (rpm)
emergency stop (rpm) Deceleration time at the time	N _{on} N _{on} N _{on} Time (s)	
	N _{sm} N _{sm} N _{sm} Time (s)	For example, $[t_{em}] = 0.05$ (s)
	Shock torque due to an emergency stop $[T_{em}]$ is equal to or less than the momentary maximum allowable torque $[T_{s2}]$	
Number of pins for reduction gear	Model Number of pins Model Number of pins (Z4) RA-20EC RA-40EA RA-80EA RA-160EA RA-160EC RA-160EC	Number of pins for RA-20E: 40
Allowable number of shock torque application times	$C_{em} = \frac{775 \times \left(\frac{T_{S2}}{T_{em}}\right)^{\frac{10}{3}}}{Z_4 \times \frac{N_{em}}{60} \times t_{ern}}$ Note $[T_{s2}]$: Momentary maximum allowable torque, refer to the rating table on page 8	$C_{\text{em}} = \frac{775 \times \left(\frac{833}{500}\right)^{\frac{10}{3}}}{40 \times \frac{15}{60} \times 0.05} = 8,497 \text{ (times)}$
rification of shock torque due to an lergency stop	Check the following condition: The allowable shock torque application count [Cem] is equal to or greater than the expected emergency stop count [Pem] If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model.	$[C_{em}]~8,497 \geq [P_{em}]~60$ According to the above condition, the tentatively selected model should be no problem.
Verify the thrust load and moment lo	ad.	
Radial load (N)	1	0 (N)
Distance to the point of radi load application (mm)	al Wi	0 (mm)
Thrust load (N)	2 2	In this example, W ₂ = W _A +W _B = (180+20×4)×9.8 = 2,548 (N) Note W _A , W _B : Refer to page 23.
Distance to the point of thru load application (mm)	st a 2	0 (mm) (As the workpiece center is located on the rotation axis)
Moment load (Nm)	$M = \frac{W_1 \times (\ell + a) + W_2 \times \ell_2}{1,000}$ a: Refer to the calculation of the tilt angle on page 33.	RA-20EA As dimension a = 63.1 (mm): $M = \frac{0 \times (0 + 63.1) + 2,548 \times 0}{1,000}$ = 0 (Nm)
rify the thrust load and moment load	Check that the thrust load and moment load are within the range in the allowable moment diagram on page 31. When radial load W ₁ is applied within dimension b, use the reduction gear within the allowable radial load. Wr: Allowable radial load, see the rating table on page 9. If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model.	For this example, Thrust load $[N_2] = 2,548$ (N) Moment load $[M] = 0$ (N) As the above values are within the range in the allowable moment diagram, the tentatively selected model should be no problem.
lect the reduction gear model that satisfi		

Select the reduction gear model that satisfies all the conditions of the above verification items. The actual reduction ratio is determined based on the motor speed, input torque, and inertia moment. Check with the motor manufacturer.

Based on the above verification result, RA-20EA is selected.

Product selection

Model code selection examples

Reduction gear selection method (2): Tentatively select a reduction gear model and evaluate the service life.

Setting/verification item	Calculation formula	Selection examples (With horizontal rotational transfer)
(1) Tentatively select a desired reduction ç		
Tentative selection of a reduction gear	Tentatively select a desired reduction gear model.	For example, tentatively select RA-20EA.
(2) Verify the maximum torque for startup	and stop.	
Verification of maximum torque for startup and stop	Check the following conditions: The allowable acceleration/deceleration torque $[T_{s1}]^{-1}$ is equal to or greater than the maximum starting torque $[T_{1}]^{-2}$ and maximum stopping torque $[T_{3}]^{-2}$ If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model. *1 $[T_{s1}]$: Refer to the rating table on page 8	$ [T_{s1}] \ 412 \ (Nm) \geq [T_1] \ 171.4 \ (Nm) $ $ [T_3] \ 162.2 \ (Nm) $ According to the above conditions, the tentatively selected model should be no problem.
	*2 [T ₁] and [T ₃]: Refer to page 25	
(3) Verify the output speed.		
N _{m0} ———— Average speed per cycle (rpm)	$N_{\text{MO}} = \frac{t_1 \times N_1 + t_2 \times N_2 + t_3 \times N_3}{t_4}$	$N_{m0} = \frac{0.5 \times 7.5 + 1.5 \times 15 + 0.5 \times 7.5}{20}$ $= 1.5 \text{ (rpm)}$
Verification of output speed	Check the following condition: The allowable output speed (100% duty ratio) [N _{s0}] ⁻¹ is equal to or greater than the average speed per cycle [N _{m0}] If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model. Contact us regarding use of the model at a speed outside the allowable output speed (40% duty ratio) [N _{s1}] ⁻¹ . Note: The value of [N _{s0}] is the speed at which the case temperature is balanced at 60°C for 30 minutes.	$[N_{s0}]~45~(rpm) \geq [N_{m0}]~1.5~(rpm)$ According to the above condition, the tentatively selected model should be no problem.
(4) 11 12 13 13 15 15 15 15 15 15 15 15 15 15 15 15 15	*1 [Nso] and [Ns1]: Refer to the rating table on page 9	
(4) Verify the shock torque at the time of a	n emergency stop.	For example, an emergency stop occurs once a month.
P _{em} Expected number of emergency stop times (times)	Based on the operation conditions.	$[P_{em}] = 1 \times 12 \times \text{required life (year) } [L_{ex}]$ = 12 x 5 = 60 (times)
T _{em} ———— Shock torque due to an emergency stop (Nm)	Load forque (Vm)	For example, [T _{em}] = 500 (Nm)
N _{em} — Speed at the time of an emergency stop (rpm)	T _{em}	For example, [N _{em}] = 15 (rpm)
t _{em} Deceleration time at the time of an emergency stop (s)	Shock torque due to an emergency stop [Tem] Set the operation conditions that meet the following requirement: Shock torque due to an emergency stop [Tem] is equal to or less than the momentary maximum allowable torque [Tsz]	For example, $[t_{em}] = 0.05$ (s)
Z ₄ — Number of pins for reduction gear	Model	Number of pins for RA-20EA: 40
C _{em} — Allowable number of shock torque application times	$C_{em} = \frac{775 \times \left(\frac{T_{S2}}{T_{em}}\right)^{\frac{10}{3}}}{Z_4 \times \frac{N_{em}}{60} \times t_{em}}$ Note $[T_{s2}]$: Momentary maximum allowable torque, refer to the rating table on page 8	$C_{\text{em}} = \frac{775 \times \left(\frac{833}{500}\right)^{\frac{10}{3}}}{40 \times \frac{15}{60} \times 0.05} = 8,497 \text{ (times)}$
Verification of shock torque due to an emergency stop	Check the following condition: The allowable shock torque application count [Cem] is equal to or greater than the expected emergency stop count [Pem]	$[C_{em}]~8.497 \geq [P_{em}]~60$ According to the above condition, the tentatively selected

Reduction gear selection method (2): Tentatively select a reduction gear model and evaluate the service life.

Setting/verification item	Calculation formula	Selection examples (With horizontal rotational transfer
) Verify the thrust load and moment load		
/ ₁ ———Radial load (N)		0 (N)
Distance to the point of radial load application (mm)	W ₁	0 (mm)
/ ₂ ——— Thrust load (N)	3	In this example, $W_2 = (180 + 20 \times 4) \times 9.8$ = 2,548 (N) Note W_A, W_B : Refer to page 23.
Distance to the point of thrust load application (mm)	W2	0 (mm) (As the workpiece center is located on the rotation axis)
Moment load (Nm)	$M = \frac{W_1 \times (\ell + a) + W_2 \times \ell_2}{1,000}$ a: Refer to the calculation of the tilt angle on page 33.	RA-20EA As dimension a = 63.1 (mm): $M = \frac{0 \times (0 + 63.1) + 2,548 \times 0}{1,000}$ = 0 (Nm)
erify the thrust load and moment load	Check that the thrust load and moment load are within the range in the allowable moment diagram on page 31. When radial load W ₁ is applied within dimension b, use the reduction gear within the allowable radial load. Wr. Allowable radial load, see the rating table on page 9. If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model.	For this example, Thrust load $[W_2] = 2,548$ (N) Moment load $[M] = 0$ (N) As the above values are within the range in the allowable moment diagram, the tentatively selected model should be no problem.
Verify the reduction gear service life.		
Life (h)	$L_h = 6,000 \times \frac{N_0}{N_m} \times \left(\frac{T_0}{T_m}\right)^{\frac{10}{3}}$	$L_{h} = 6,000 \times \frac{15}{12} \times \left(\frac{167}{110.1}\right)^{\frac{10}{3}}$ = 30,072 (h)
Yacy — Number of cycles per day (times)	$Q_{1cy} = \frac{Q_1 \times 60 \times 60}{t_4}$	$Q_{1cy} = \frac{12 \times 60 \times 60}{20} = 2,160 \text{ (times)}$
Operating hours per day (h)	$Q_3 = \frac{Q_{1\text{cy}} \times (t_1 + t_2 + t_3)}{60 \times 60}$	$Q_3 = \frac{2,160 \times (0.5+1.5+0.5)}{60 \times 60} = 1.5 \text{ (h)}$
4 — Operating hours per year (h)	$Q_4 = Q_3 \times Q_2$	Q ₄ = 1.5 × 365 = 548 (h)
rear — Reduction gear service life (year)	$L_{year} = \frac{L_{h}}{Q_{4}}$	$L_{year} = \frac{30,072}{548} = 54.9 \text{ (year)}$
ex Required life (year)	Based on the operation conditions	5 years
erification of the service life	Check the following condition: $[L_{ex}]$ is equal to or less than $[L_{year}]$ If the tentatively selected reduction gear is outside of the specifications, change the reduction gear model.	$[L_{\text{ex}}] \ 5 \ (\text{year}) \le [L_{\text{year}}] \ 54.9 \ (\text{year})$ According to the above condition, the tentatively selected model should be no problem.
	all the conditions of the above verification items.	Based on the above verification result, RA-20EA is

Product selection

Model code selection examples

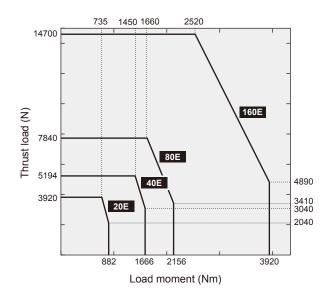
Limitation on the motor torque

A limitation is imposed on the motor torque value so that the shock torque applied to the reduction gear does not exceed the momentary maximum allowable torque.

Setting/verification item	Calculation formula	Selection examples (With horizontal rotational transfer)
T _{M1} Motor momentary maximum torque (Nm)	Determine based on the motor specifications.	For example, T _{M1} = 10 (Nm)
Maximum torque generated at the TM10UT — output shaft for the reduction gear (Nm)	$T_{M1out} = T_{M1} \times R \times \frac{100}{\eta}$	For example, calculate the maximum torque generated at the output shaft for the reduction gear based on the specifications when RA-20EA-160 was selected.
(When an external shock is applied at the time of an emergency stop or motor stop)	R: Actual reduction ratio $\eta: Startup \ \text{efficiency (\%), refer to the rating table on page 8}$	$T_{M \text{ 1out}} = 10 \times 160 \times \frac{100}{75}$ = 2,133 (Nm)
Maximum torque generated at the TM20UT — output shaft for the reduction gear (Nm) (When a shock is applied to the output shaft due to hitting by an obstacle)	$T_{M2out} = T_{M1} \times R \times \frac{\eta}{100}$	$T_{M2out} = 10 \times 160 \times \frac{75}{100}$ = 1,200 (Nm)
Limitation on motor torque value	Check the following condition: The momentary maximum allowable torque [Ts2] ⁻¹ is equal to or greater than the maximum torque generated at the output shaft for the reduction gear [TM10url] and [TM20url] If the above condition is not satisfied, a limitation is imposed on the maximum torque value of the motor. *1 [Ts2]: Refer to the rating table on page 8	[Ts₂] 833 (Nm) ≤ [T _{M1} out] 2,133 (Nm) and [T _{M2} out] 1,200 (Nm) According to the above condition, the torque limit is set for the motor.

Product selection

Allowable moment diagram



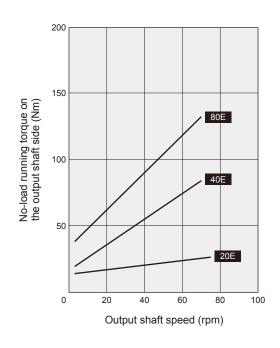
Technical data

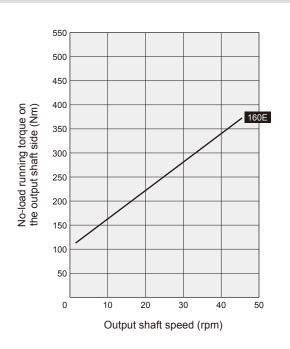
No-load running torque

RA Series

Case temperature: Lubricant:

30°C Grease (Molywhite RE00)





The no-load running torque that is converted to the input shaft side value should be calculated using the following equation:

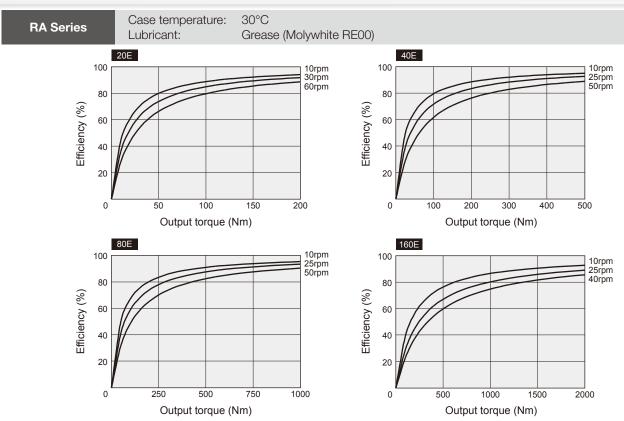
No-load running torque on the input shaft side (Nm)

No-load running torque on the output shaft side (Nm)

Speed ratio

Technical data

Efficiency table



Technical data

Low-temperature characteristics

RA Series

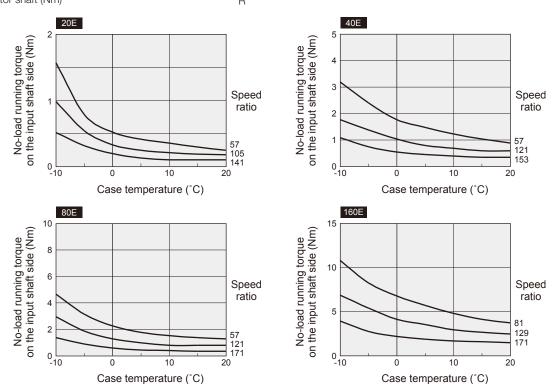
Input speed: 2000 rpm

Lubricant: Grease (Molywhite RE00)

When the reduction gear is used at a low temperature, viscosity of lubricant increases and causes a larger no-load running torque. The no-load running torque at low temperature is shown below.

Use the following formula to calculate the no-load running torque converted to the motor shaft.

No-load running torque converted = $\frac{\text{Torque converted into the output shaft (Nm)}}{\text{R}}$ (R: speed ratio value)



Technical data

Calculation of tilt angle and torsion angle

Calculation of tilt angle

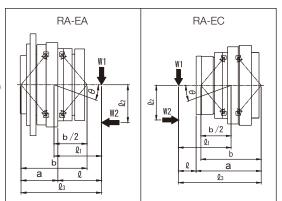
When a load moment occurs with an external load applied, the output shaft will tilt in proportion to the load moment (If ℓ_3 is larger than b) The moment rigidity indicates the rigidity of the main bearing, and it is represented by the load moment value required for tilting the main bearing by 1 arc.min.

θ : Tilt angle of the output shaft (arc.min.)
 M₁ : Moment rigidity (Nm/arc.min.)

$$\theta = \begin{array}{c} \frac{W_1\ell_1 + W_2\ell_2}{M_1 \times 10^3} \\ \end{array} \begin{array}{c} M_1 \\ \ell_1, \ \ell_2 \end{array} \begin{array}{c} \text{: Moment rigidity (Nm/arc.min.} \\ \text{: Load (N)} \\ \text{: Distance to the point of load application (mm)} \end{array}$$

 ℓ_1 : $\ell + \frac{b}{2} + a - b$

Distance from the output shaft installation surface to the point of load application (mm)



	Moment rigidity	Dimensio	ns (mm)
Model	(central value) (Nm/arc.min.)	а	b
RA-20EA	372	63.1	113.3
RA-40EA	931	83.1	143.7
RA-80EA	1,176	81.5	166
RA-160EA	2,940	93.8	210.9

	Moment rigidity	Dimensio	ons (mm)
Model	(central value) (Nm/arc.min.)	а	b
RA-20EC	372	122.2	113.3
RA-40EC	931	148.1	143.7
RA-80EC	1,176	158.4	166
RA-160EC	2,940	201.8	210.9

Calculation of torsion angle

Calculate the torsion angle when the torque is applied in a single direction, using an example of RA-160E.

1) When the load torque is 30 Nm.....Torsion angle (ST₁) When the load torque is within the lost motion range

$$ST_1 = \frac{30}{47} \times \frac{1 \text{ (arc.min.)}}{2} = 0.32 \text{arc.min. or less}$$

2) When the load torque is 1,300 Nm.....Torsion angle (ST₂)

When the load torque is within the rated torque range

$$ST_2 = \frac{1}{2} + \frac{1300 - 47.0}{392} = 3.70 arc.min.$$

Note: 1. The torsion angles that are calculated above are for a single reduction gear.

2. Contact us for the customized specifications for lost motion.

	Torsional rigidity	Lost n	notion	
Model	(central value) (Nm/arc.min.)	Lost motion (arc.min.)	Measured torque (Nm)	Backlash (arc.min.)
RA-20E	49		± 5.0	
RA-40E	108	1.0	± 12.3	1.0
RA-80E	196	1.0	± 23.5	1.0
RA-160E	392		± 47.0	

Design points

Reduction gear installation components

Installation of the reduction gear and mounting it to the output shaft

When installing the reduction gear and mounting it to the output shaft, use hexagon socket head cap screws and tighten them at the torque as specified below. This is to satisfy the momentary maximum allowable torque, which is noted in the rating table.

The use of the serrated lock washers are recommended to prevent the hexagon socket head cap screws from loosening and to protect the seat surface from flaws.

• Hexagon socket head cap screw

<Bolt tightening torque and tightening force>

Hexagon socket head cap screw nominal size x pitch (mm)	Tightening torque (Nm)	Tightening force F (N)	Bolt specification
. ,	, ,		
M5 × 0.8	9.01 ± 0.49	9,310	Hexagon socket head cap screw
M6 × 1.0	15.6 ± 0.78	13,180	JIS B 1176: 2006
M8 × 1.25	37.2 ± 1.86	23,960	Strength class
M10 × 1.5	73.5 ± 3.43	38,080	JIS B 1051: 2000 12.9
M12 × 1.75	129 ± 6.37	55,100	Thread
M16 × 2.0	319 ± 15.9	103,410	JIS B 0209: 2001 6g

Note: 1. The tightening torque values listed are for steel or cast iron material.

If softer material, such as aluminum or stainless, is used, limit the tightening torque. Also take the transmission torque and load moment into due consideration.

<Calculation of allowable transmission torque of bolts>

$T = F \times \mu \times \frac{D}{2 \times 1,000} \times n$	Т	Allowable transmission torque by tightening bolt (Nm)
	F	Bolt tightening force (N)
	D	Bolt mounting P.C.D. (mm)
	μ	Friction factor
		μ=0.15: When lubricant remains on the mating face.
		μ=0.20: When lubricant is removed from the mating face.
	n	Number of bolts (pcs.)

Serrated lock washer for hexagon socket head cap screw

(Unit: mm)

Name: Belleville spring washer (made by Heiwa Hatsujyo Industry Co., Ltd.)

Corporation symbol: CDW-H

CDW-L (Only for M5)

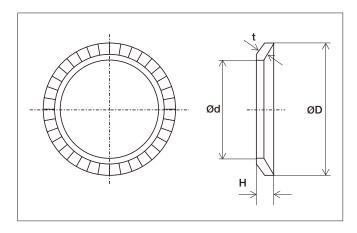
2.3

Material: S50C to S70C Hardness: HRC40 to 48

16

16.9

Nominal	Belle	OD of eville washer			
size	Ød	ØD	t	Н	
5	5.25	8.5	0.6	0.85	
6	6.4	10	1.0	1.25	
8	8.4	13	1.2	1.55	
10	10.6	16	1.5	1.9	
12	12.6	18	1.8	2.2	



Note: When using any equivalent washer, select it with special care given to its outside diameter D.

2.8

Lubrication

- The standard lubrication method for the RA reduction gears is greasing.
 - Before the reduction gear is shipped, it is filled with our recommended grease (Molywhite RE00).
 - When operating a reduction gear filled with the appropriate amount of grease, the standard replacement time due to deterioration of the grease is 20,000 hours.
 - When using the gear with deteriorated grease or under an inappropriate ambient temperature condition (40°C or more), check the deterioration condition of the grease and determine the appropriate replacement cycle.
- Specified grease name

Grease name	Molywhite RE00
Manufacturer	Nabtesco Corporation
Ambient temperature	-10 to 40°C

• It is recommended that the running-in operation is performed.

Abnormal noise or torque variation may occur during operation due to the characteristics of the lubricant. There is no problem with the quality when the symptom disappears after the running-in operation is performed.

Requirements for equipment design

If the lubricant leaks from the reduction gear or if the motor fails, the reduction gear must be removed. Design the equipment while taking this into consideration.

Inertia moment calculation formula

Shape	I(kgm²)	Shape	I(kgm²)
1. Cylinder solid M(kg) Z R(m) a(m)	$I_x = \frac{1}{2} M R^2$ $I_y = \frac{1}{4} M \left(R^2 + \frac{a^2}{3} \right)$ $I_z = I_y$	6. Horizontal movement by conveyor M1(kg) V(m/min) R(m) R(m) N(rpm)	$I = \left(\frac{M_1 + M_2}{2} + M_3 + M_4\right) \times R^2$
2. Cylinder hollow M(kg) Z Rt(m) Rz(m) Rz(m	$I_{x} = \frac{1}{2}M \left(R_{1}^{2} + R_{2}^{2}\right)$ $I_{y} = \frac{1}{4}M \left\{ \left(R_{1}^{2} + R_{2}^{2}\right) + \frac{a^{2}}{3}\right\}$ $I_{z} = I_{y}$	7. Horizontal movement by lead screw V(m/min) N(rpm) Lead: P(m/rev)	$I = \frac{M}{4} \left(\frac{V}{\pi \times N} \right)^2 = \frac{M}{4} \left(\frac{P}{\pi} \right)^2$
3. Oval cross section M(kg) Z X a(m) Z E b(m)	$I_{x} = \frac{1}{16} M \left(b^{2} + c^{2} \right)$ $I_{y} = \frac{1}{4} M \left(\frac{c^{2}}{4} + \frac{a^{2}}{3} \right)$ $I_{z} = \frac{1}{4} M \left(\frac{b^{2}}{4} + \frac{a^{2}}{3} \right)$	8. Up/down movement by hoist Mz(kg) N(rpm) V(m/min) M1(kg)	$I = M_1 R^2 + \frac{1}{2} M_2 R^2$
4. Rectangle M(kg) Z X A(m) A(m) A(m) Z A(m) A(m) A(m)	$I_{x} = \frac{1}{12} M (b^{2} + c^{2})$ $I_{y} = \frac{1}{12} M (a^{2} + c^{2})$ $I_{z} = \frac{1}{12} M (a^{2} + b^{2})$	9. Parallel axis theorem M(kg) Center axis n(m) Rotation axis	I = I ₀ + Mη ² I ₀ : Moment of inertia of any object about an axis through its center of mass I: Moment of inertia about any axis parallel to the axis through its center of mass η: Perpendicular distance between the above two axes
5. General application M(kg) V(m/min) R(m) N(rpm)	$I = \frac{M}{4} \left(\frac{V}{\pi \times N} \right)^2 = MR^2$		

Troubleshooting checksheet

Check the following items in the case of trouble like abnormal noise, vibration, or malfunctions.

When it is not possible to resolve an abnormality even after verifying the corresponding checkpoint, obtain a "Reduction Gear Investigation Request Sheet" from the download page in our Website, fill in the necessary information, and contact

our Customer Support Center at Tsu Plant.

[URL]: http://precision.nabtesco.com/

The trouble started immediately after installation of the reduction gear

Checked	Checkpoint
	Make sure the equipment's drive section (the motor side or the reduction gear output surface side) is not interfering with another component.
	Make sure the equipment is not under a greater than expected load (torque, moment load, thrust load).
	Make sure the required number of bolts are tightened uniformly with the specified tightening torque.
	Make sure the reduction gear, motor, or your company's components are not installed at a slant.
	Make sure the specified amount of Nabtesco-specified lubricant has been added.
	Make sure there are no problems with the motor's parameter settings.
	Make sure there are no components resonating in unity.
	Make sure the input gear is appropriately installed on the motor.
	Make sure there is no damage to the surface of the input gear teeth.
	Make sure the input gear specifications (precision, number of teeth, module, shift coefficient, dimensions of each part) are correct.
	Make sure the flange and other components are designed and manufactured with the correct tolerances.

The trouble started during operation

Checked	Checkpoint	
	Make sure the equipment has not been in operation longer than the calculated service life.	
	Make sure the surface temperature of the reduction gear is not higher than normal during operation.	
	Make sure the operation conditions have not been changed.	
	Make sure there are no loose or missing bolts.	
	Make sure the equipment is not under a greater than expected load (torque, moment load, thrust load).	
	Make sure the equipment's drive section is not interfering with another component.	
	Make sure an oil leak is not causing a drop in the amount of lubricant.	
	Make sure there are no external contaminants in the gear, such as moisture or metal powder.	
	Make sure no lubricant other than that specified is being used.	

▶ Area In North and South America / In Europe and Africa / In Asia and others / Germany: FAX USA: / Osaka Sales Office: 1-248-553-3070 / 49-211-364677 / 81-6-6341-7182

Order Information Sheet (Please complete the form below)

Date.

Company Name:	Dept. Name:
Name:	E-mail:

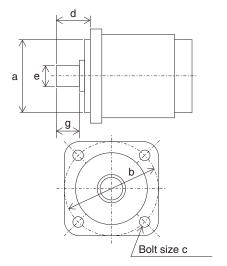
FAX.

◆ System configuration and selected motor

We would appreciate if you could provide your system configuration drawing that helps us to understand the speed, constant torque, and load inertia moment of the output shaft for the reduction gear.

System configuration

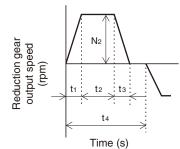
TEL.



Motor model		а	Motor mounting pilot diameter (mm)
Р	Motor rated output (KW)	b	Motor mounting bolt P.C.D (mm)
Тмо	Motor rated torque (Nm)	С	Motor mounting bolt size (mm)
T _{M1}	Motor momentary maximum torque (Nm)	d	Motor shaft length (mm)
N мо	Motor rated speed (rpm)	е	Motor shaft diameter (mm)
		g	Motor shaft effective length (mm)

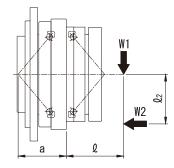
◆ Operation pattern (output shaft for the reduction gear)

Please inform us of whether a key is attached or not, and when it is attached, also inform us of its dimensions.

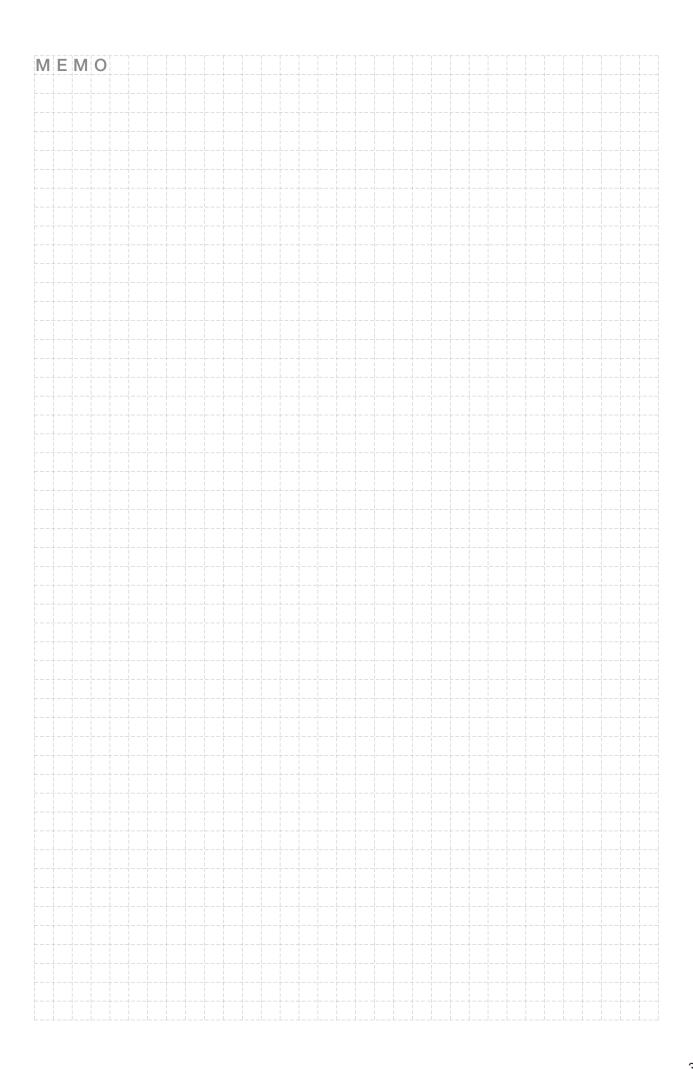


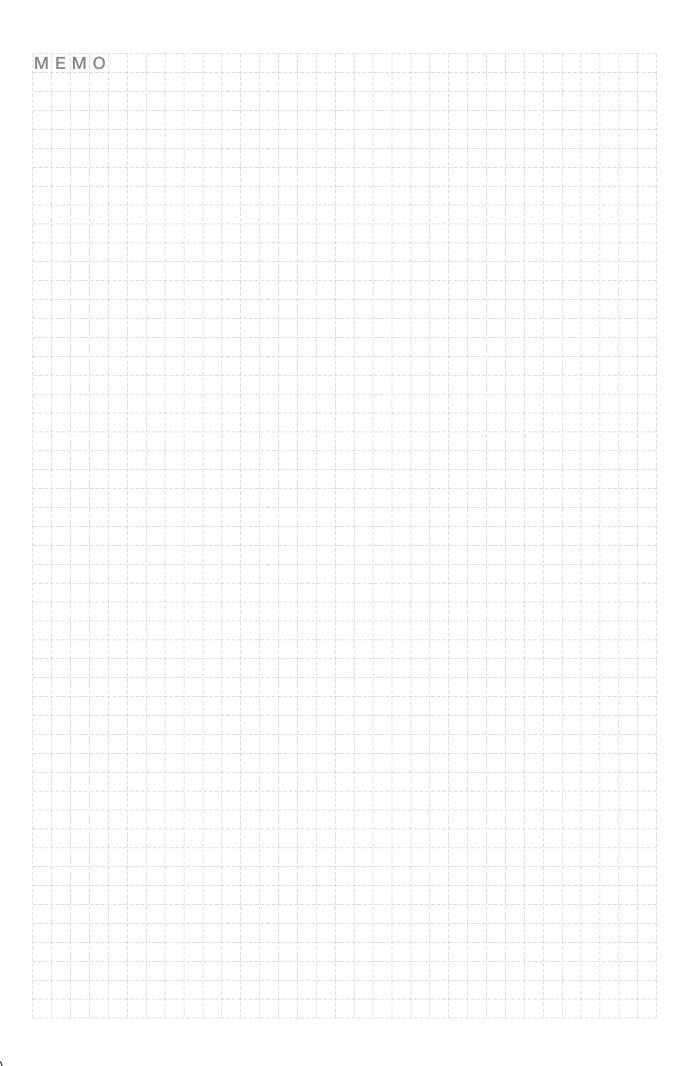
	* *	
t ₁	Acceleration time (s)	
t 2	Constant speed operation time (s)	
t ₃	Deceleration time (s)	
t ₄	One operation cycle time (s)	
Q _{1CY}	Number of operation cycles per day (times)	
Q ₂	Number of operating days per year (days)	
N ₂	Constant speed (rpm)	
T 1	Max. torque for startup (Nm)	
T 2	Constant torque (Nm)	
T 3	Max. torque for stop (Nm)	

◆ External load (output shaft for the reduction gear)



W 1	Radial load (N)	
Q	Distance to the point of radial load application (mm)	
W ₂	Axial load (N)	
Q ₂	Distance to the point of axial load application (mm)	





Warranty

- 1. In the case where Nabtesco confirms that a defect of the Product was caused due to Nabtesco's design or manufacture within the Warranty Period of the Product, Nabtesco shall repair or replace such defective Product at its cost. The Warranty Period shall be from the delivery of the Product by Nabtesco or its distributor to you ("Customer") until the end of one (1) year thereafter, or the end of two thousand (2,000) hours from the initial operation of Customer's equipment incorporating the Product at end user's production line, whichever comes earlier.
- 2. Unless otherwise expressly agreed between the parties in writing, the warranty obligations for the Product shall be limited to the repair or replacement set forth herein. OTHER THAN AS PROVIDED HEREIN, THERE ARE NO WARRATIES ON THE PRODUCT, EXPRESS OR IMPLIED, INCLUDING WITHOUT LIMITATION ANY IMPLIED WARRANTY OF MERCHANTABILITY OR FITNESS FOR A PARTICULAR PURPOSE.
- 3. The warranty obligation under the Section 1 above shall not apply if:
 - a) the defect was caused due to the use of the Product deviated from the Specifications or the working conditions provided by Nabtesco;
 - b) the defect was caused due to exposure to foreign substances or contamination (dirt, sand etc.)
 - c) lubricant or spare part other than the ones recommended by Nabtesco was used in the Product;
 - d) the Product was used in an unusual environment (such as high temperature, high humidity, a lot of dust, corrosive/volatile/inflammable gas, pressurized/depressurized air, under water/liquid or others except for those expressly stated in the Specifications);
 - e) the Product was disassembled, re-assembled, repaired or modified by anyone other than Nabtesco;
 - f) the defect was caused due to the equipment into which the Product was installed;
 - g) the defect was caused due to an accident such as fire, earthquake, lightning, flood or others; or
 - h) the defect was due to any cause other than the design or manufacturing of the Product.
- 4. The warranty period for the repaired/replaced Product/part under the Section 1 above shall be the rest of the initial Warranty Period of the defective Product subjected to such repair/replace.

Nabtesco Corporation

China

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Osaka Sales Office

